

**Homework Exercises for Chapter 15. The Linear Plane Stress Triangle**  
**Solutions**

**EXERCISE 15.1** From (15.21):

$$\mathbf{K}^e = \mathbf{B}^T \mathbf{E} \mathbf{B} \int_{\Omega^e} (h_1 \zeta_1 + h_2 \zeta_2 + h_3 \zeta_3) d\Omega^e. \quad (\text{E15.6})$$

From (15.27):

$$\int_{\Omega^e} \zeta_1 d\Omega^e = \int_{\Omega^e} \zeta_2 d\Omega^e = \int_{\Omega^e} \zeta_3 d\Omega^e = \frac{A}{3}. \quad (\text{E15.7})$$

Replacing we get

$$\mathbf{K}^e = Ah_m \mathbf{B}^T \mathbf{E} \mathbf{B}, \quad (\text{E15.8})$$

where  $h_m = (h_1 + h_2 + h_3)/3$  is the mean thickness. This also happens to be the thickness at the centroid.

**EXERCISE 15.2** Tabulation of the right-hand side of (15.27) for  $i + j + k \leq 3$ :

$i$	$j$	$k$	$\frac{i! j! k!}{(i + j + k + 2)!}$
0	0	0	1/2
1	0	0	1/6
2	0	0	1/12
1	1	0	1/24
3	0	0	1/20
2	1	0	1/60
1	1	1	1/120

**EXERCISE 15.3** Use (15.27). The integral with monomial exponents  $i = 3, j = k = 0$  evaluates to  $6A/60$ ; that with  $i = 2, j = 1, k = 0$  to  $2A/60$ ; and that with  $i = j = k = 1$  to  $A/60$ . The  $x$  forces vanish:  $f_{xi} = 0, i = 1, 2, 3$ . For the  $y$  forces:

$$f_{y1} = \frac{A}{60} [b_{y1}(6h_1 + 2h_2 + 2h_3) + b_{y2}(2h_1 + 2h_2 + h_3) + b_{y3}(2h_1 + h_2 + 2h_3)] \quad (\text{E15.9})$$

$f_{y2}$  and  $f_{y3}$  follow by 3-cyclic permutation. Verification for  $b_{y1} = b_{y2} = b_{y3} = b_y$  and  $h_1 = h_2 = h_3 = h$  gives  $hAb_y/3$  at the three corners, which is the element-by-element lumping.

**EXERCISE 15.4**

$$\begin{aligned} f_{x1} &= \frac{1}{6}h L_{21}(2q_{x1} + q_{x2}), & f_{x2} &= \frac{1}{6}h L_{21}(q_{x1} + 2q_{x2}), \\ f_{y1} &= \frac{1}{6}h L_{21}(2q_{y1} + q_{y2}), & f_{y2} &= \frac{1}{6}h L_{21}(q_{y1} + 2q_{y2}), & f_{x3} &= f_{y3} = 0. \end{aligned} \quad (\text{E15.10})$$

Here  $L_{21} = \sqrt{x_{21}^2 + y_{21}^2}$  is the length of side 1-2.

**EXERCISE 15.5**

$$\mathbf{K}^e = \begin{bmatrix} 18.75 & 9.375 & -12.5 & -6.25 & -6.25 & -3.125 \\ 9.375 & 18.75 & 6.25 & 12.5 & -15.625 & -31.25 \\ -12.5 & 6.25 & 75.0 & -37.5 & -62.5 & 31.25 \\ -6.25 & 12.5 & -37.5 & 75.0 & 43.75 & -87.5 \\ -6.25 & -15.625 & -62.5 & 43.75 & 68.75 & -28.125 \\ -3.125 & -31.25 & 31.25 & -87.5 & -28.125 & 118.75 \end{bmatrix} \quad (\text{E15.11})$$

**EXERCISE 15.6** Set  $u_{x1} = u_{x2} = u_{x3} = 1$  and  $u_{y1} = u_{y2} = u_{y3} = 0$ . This is a translational rigid body motion along  $x$  and consequently the element node forces obtained from  $\mathbf{f}^e = \mathbf{K}^e \mathbf{u}^e$  must be exactly zero. These forces are the sum of columns 1, 3 and 5. A similar result is obtained for columns 2, 4 and 6 on switching  $x$  and  $y$ . Because the stiffness matrix is symmetric, the same result holds on replacing “columns” by “rows.” (The result extends to any mechanical finite element.)

**EXERCISE 15.7** Not assigned.

**EXERCISE 15.8** Not assigned.

**EXERCISE 15.9** Not assigned.

**EXERCISE 15.10** Not assigned.

**EXERCISE 15.11** Not assigned

**EXERCISE 15.12** The answer should be a hierarchical diagram such as

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Main program in Cell 8 - drives the FEM analysis
  GenerateNodes - generates node coordinates of regular mesh
  GenerateTriangles - generate element node lists of regular mesh
  GenerateEndAxialForces - generates end axial forces on cantilever
  GenerateEndMomentForces - generates end moment forces on cantilever
  GenerateEndShearForces - generates end-shear loads on cantilever
  Plot2DMesh - plots a 2D mesh
  LinearSolutionOfPlaneStressModel - drives solution of FEM problem
    AssembleMasterStiffOfPlaneStressModel - assembles master stiffness
      StiffnessOf3NodePlaneStressTriangle - forms element stiffness
        MergeElemIntoMasterStiff - merges element stiffness into master
          stiffness
      ModifyNodeForcesForDBC - modifies node forces for displacement BC
      ModifyMasterStiff - modifies master stiffness for displacement BC
      (Linear solution: done by Mathematica built-in Inverse function)
      StressesInPlaneStressModel - recovers element stresses
  ContourPlotNodeFuncOver2DMesh - Contourplots a node-defined
    function over 2D mesh
    PlotFunctionOverTriangle - plot function over triangle
      ContourPolyColor - picks polygon display color
    PlotFunctionOverQuadrilateral - like title says
      ContourPolyColor - picks polygon display color
  ContourPlotElemFuncOver2DMesh - Contourplots an element-defined
    function over 2D mesh
    PlotFunctionOverTriangle - plot function over triangle
      ContourPolyColor - picks polygon display color
    PlotFunctionOverQuadrilateral - like title says
      ContourPolyColor - picks polygon display color

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