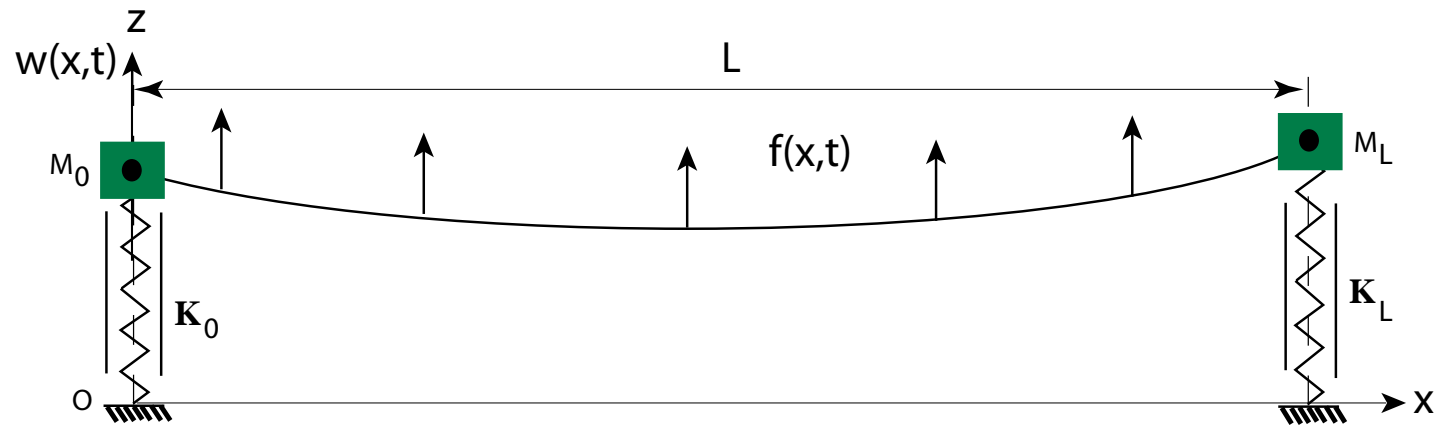


ASEN 5022 - Spring 2005

Dynamics of Aerospace Structures

Lecture 10: 17 February

Cables with End Masses and End Springs



Cables with End Masses and End Springs

Formulation via Hamilton's principle:

$$\int_{t_1}^{t_2} [\delta L + \delta \bar{W}_{noncons}] dt = 0, \quad L = T - V$$

$$T = \int_0^L \frac{1}{2} \rho(x) \dot{w}^2(x, t) dx + \frac{1}{2} M_0 \dot{w}^2(0, t) + \frac{1}{2} M_L \dot{w}^2(L, t) \quad (1)$$

$$V = \int_0^L \frac{1}{2} T(x) w_x^2(x, t) dx + \frac{1}{2} K_0 w^2(0, t) + \frac{1}{2} K_L w^2(L, t)$$

Symbol Definitions:

Transverse displacement: $w(x, t)$ (m)

Time derivative: $\dot{w} = \frac{\partial w}{\partial t}$ (m/s)

Spatial derivative: $w_x = \frac{\partial w}{\partial x}$ (m/m)

String tension: $T(x)$ (N)

Mass per unit cable length: $\rho(x)$ (kg/m)

End springs: (K_0, K_L) (N/m)

End masses: (M_0, M_L) (N/m)

Evaluation of $\int_{t_1}^{t_2} \delta T dt$

$$\begin{aligned} \int_{t_1}^{t_2} \delta T dt = & \\ - \int_{t_1}^{t_2} \{ & \left[\int_0^L \rho(x) \ddot{w}(x, t) \delta w(x, t) dx \right] \\ & + M_0 \ddot{w}(0, t) \delta w(0, t) \\ & + M_L \ddot{w}(L, t) \delta w(L, t) \} dt \end{aligned} \quad (2)$$

Evaluation of δV

$$\delta V =$$

$$- \int_0^L T(x) w_{xx}(x, t) \delta w(x, t) dx$$

$$+ [T(x) w_x(x, t) \delta w(x, t)]_{x=L}$$

$$- [T(x) w_x(x, t) \delta w(x, t)]_{x=0}$$

$$+ [K_L w(x, t) \delta w(x, t)]_{x=L}$$

$$+ [K_0 w(x, t) \delta w(x, t)]_{x=0}$$

(3)

Virtual work due to nonconservative force $f(x, t)$

$$\delta \bar{W}_{noncons} = \int_0^L f(x, t) \delta w(x, t) dx \quad (4)$$

Substituting (4), (3) and (2) into (1), one obtains Hamilton's principle:

$$\int_{t_1}^{t_2} [\delta(T - V) + \delta \bar{W}_{noncons}] dt = 0 \quad (1)$$

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Hamilton's principle for cable with end masses and end springs

$$\begin{aligned} & \int_{t_1}^{t_2} \left\langle \int_0^L \{ [-\rho(x) \ddot{w}(x, t) \right. \\ & + T(x) w_{xx}(x, t) + f(x, t)] \delta w(x, t) \} dx \\ & - \{ [(T(x) w_x(x, t) + K_L w(x, t) \\ & + M_L \ddot{w}(x, t))] \delta w(x, t) \}_{x=L} \\ & - \{ [-T(x) w_x(x, t) + K_L w(x, t) \\ & + M_0 \ddot{w}(x, t)] \delta w(x, t) \}_{x=0} \rangle dt = 0 \end{aligned} \tag{5}$$

The governing equation of motion:

$$\rho(x)\ddot{w}(x, t) = T(x) w_{xx}(x, t) + f(x, t) \quad (6)$$

The boundary conditions:

$$\{[(T(x) w_x(x, t) + K_L w(x, t) + M_L \ddot{w}(x, t)]\delta w(x, t)\}_{x=L} = 0 \quad (7)$$

$$\{[-T(x) w_x(x, t) + K_0 w(x, t) + M_0 \ddot{w}(x, t)]\delta w(x, t)\}_{x=0} = 0 \quad (8)$$

Observations:

- 1. The governing equation of motion for the case of end masses and end springs is the same as the case without**

any end conditions. This means the governing equation of motion is the same for all possible boundary conditions.

2. Therefore, the general form of solution

$$w(x, t) = F(t) [C_1 \sin \beta x + C_2 \cos \beta x] \quad (9)$$

is applicable to all the cable vibration problems.

Boundary conditions at $x = L$: From (7) we find

$$T(L) w_x(L, t) + K_L w(L, t) + M_L \ddot{w}(L, t) = 0 \quad (10)$$

$$\mathbf{or} \quad w(L, t) = 0 \quad (11)$$

Boundary conditions at $x = 0$: From (8) we find

$$-T(0) w_x(0, t) + K_0 w(0, t) + M_0 \ddot{w}(0, t) = 0 \quad (12)$$

$$\mathbf{or} \quad w(0, t) = 0 \quad (13)$$

From (10)-(13), one finds that the two essential boundary conditions given by (11) and (13) are special cases of the two natural boundary conditions given by (10) and (12), since the former two are obtained from the latter two if $K_L \rightarrow \infty$ and $K_0 \rightarrow \infty$, respectively.

Substituting (9) into (10) and (12), together with $F(t) = \bar{F} e^{j\omega t}$, yields the following characteristic equation:

$$\begin{bmatrix} \bar{\beta} \cos \bar{\beta} + & -\bar{\beta} \sin \bar{\beta} + \\ (\kappa_L - \mu_L \bar{\beta}^2) \sin \bar{\beta} & (\kappa_L - \mu_L \bar{\beta}^2) \cos \bar{\beta} \\ -\bar{\beta} & (\kappa_0 - \mu_0 \bar{\beta}^2) \end{bmatrix} \begin{Bmatrix} C_1 \\ C_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \quad (14)$$

$$\begin{aligned} \beta &= \omega \sqrt{\frac{\rho}{T}}, & \bar{\beta} &= \beta L \\ \kappa_L &= \frac{K_L L}{T}, & \kappa_0 &= \frac{K_0 L}{T} \\ \mu_L &= \frac{M_L}{\rho L}, & \mu_0 &= \frac{M_0}{\rho L} \end{aligned} \quad (15)$$

The characteristic equation of (14) is found by taking the determinant to be zero:

$$\begin{aligned} & (\kappa_0 - \mu_0 \bar{\beta}^2)(\kappa_L - \mu_L \bar{\beta}^2) \sin \bar{\beta} \\ & + \bar{\beta} \cos \bar{\beta} [(\kappa_0 - \mu_0 \bar{\beta}^2) + (\kappa_L - \mu_L \bar{\beta}^2)] \quad (16) \\ & - \bar{\beta}^2 \sin \bar{\beta} = 0 \end{aligned}$$

We now examine several special cases in the following.

1. *Fixed-Fixed Ends* ($k_L \rightarrow \infty, k_0 \rightarrow \infty$)

This case corresponds to ($k_L \rightarrow \infty, k_0 \rightarrow \infty$) so that, by dividing (16) by ($k_L k_0$) yields:

$$\sin \bar{\beta} = \sin \beta L = 0 \quad \Rightarrow \quad \beta L = k\pi, k = 1, 2, \dots$$

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$$\omega_k = \frac{k\pi}{L} \sqrt{\frac{T}{\rho}}, \quad k = 1, 2, 3, \dots$$

(17)

which was already discussed previously.

2. *Fixed-Free Ends* ($k_L \rightarrow 0$, $k_0 \rightarrow \infty$, $\mu_L = \mu_0 = 0$)

This case leads to the following characteristic equation from (16):

$$\begin{aligned}\cos \bar{\beta} &= \sin \beta L = 0 \\ \beta L &= \frac{(2k - 1)}{2} \pi, \quad k = 1, 2, \dots \\ \Downarrow & \\ \omega_k &= \frac{(2k - 1)\pi}{2L} \sqrt{\frac{T}{\rho}}, \quad k = 1, 2, 3, \dots\end{aligned}\tag{18}$$

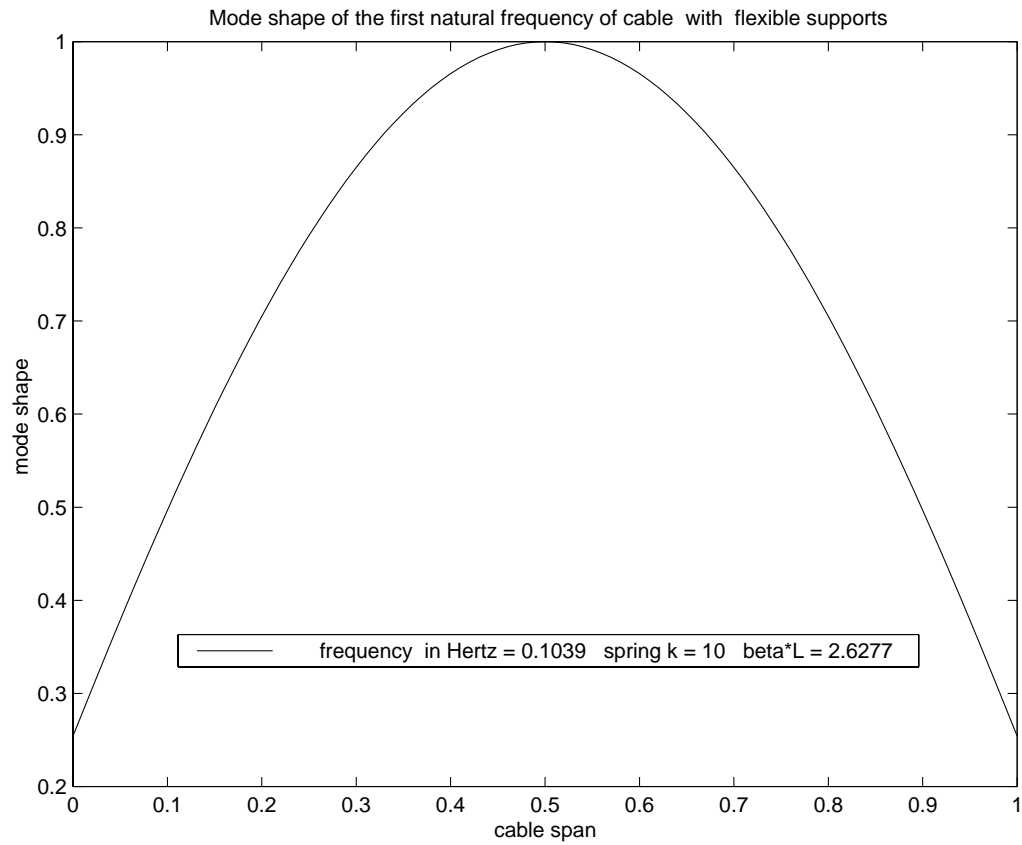
which was also treated before.

3. Flexible Supports ($k_L = k_0 \neq 0, \mu_L = \mu_0 = 0$)

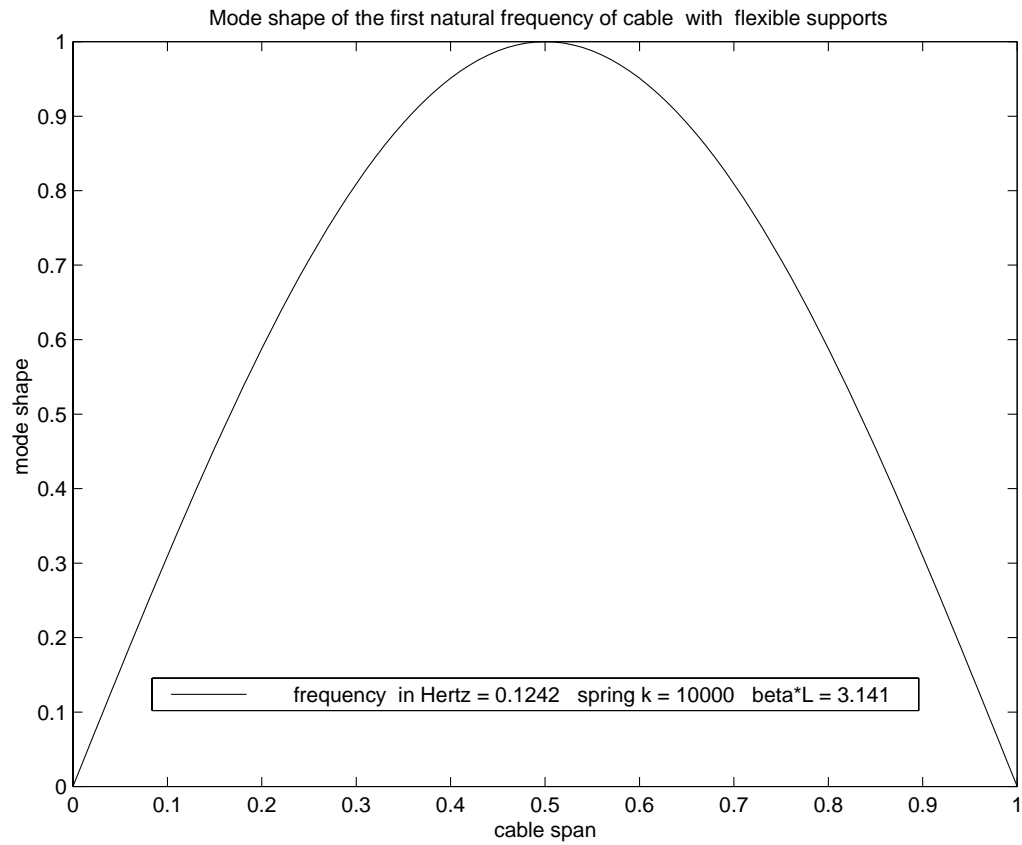
This case leads to the following characteristic equation from (16):

$$(k^2 - \bar{\beta}^2) \tan \bar{\beta} + 2k\bar{\beta} = 0, \quad k = k_L = k_0 \quad (19)$$

Other general cases can be evaluated using (17). This is left for exercises



Mode Shapes of Cable with End Springs



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