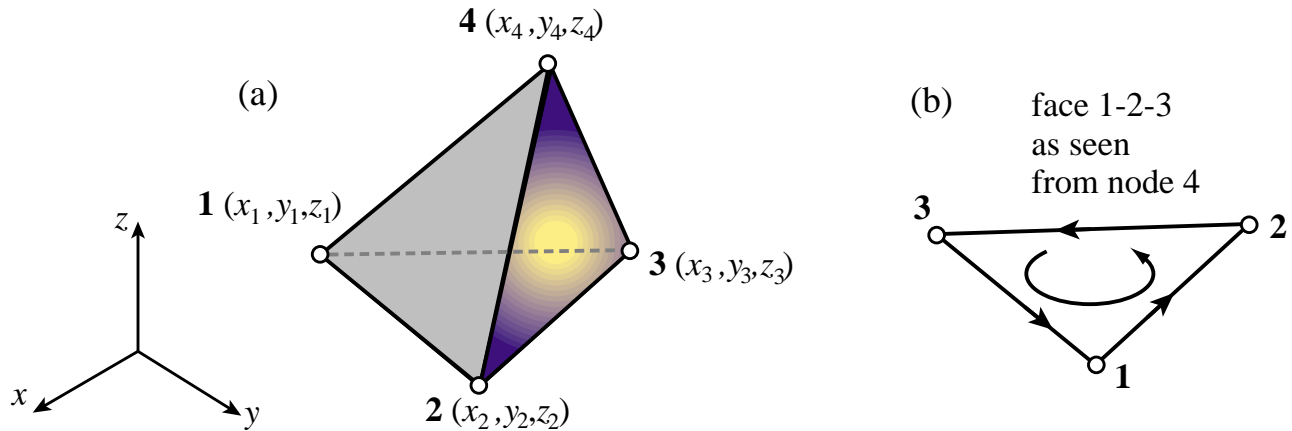


# 16

## The Linear Tetrahedron

# The Linear (4-Node) Tetrahedron

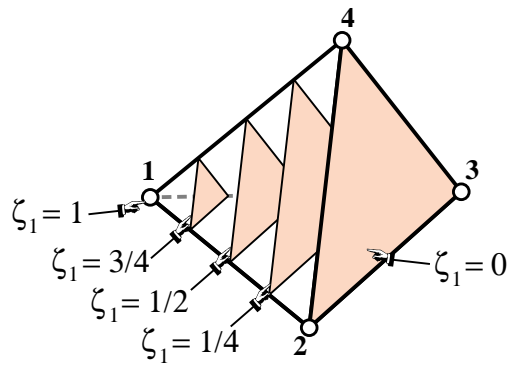


Volume of tetrahedron is

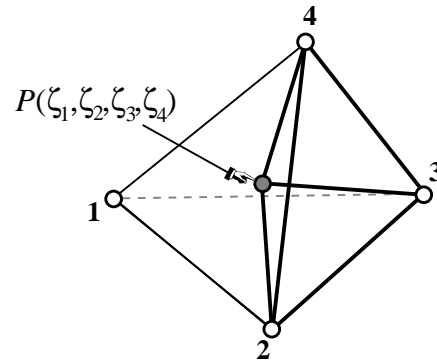
$$V = \frac{1}{6} \det \begin{bmatrix} 1 & 1 & 1 & 1 \\ x_1 & x_2 & x_3 & x_4 \\ y_1 & y_2 & y_3 & y_4 \\ z_1 & z_2 & z_3 & z_4 \end{bmatrix}$$

This is a signed quantity and must be  $>0$

## Tetrahedron Natural Coordinates



interpretation as a  
moving surface



interpretation as  
volume ratio

## Tetrahedron Coordinates (cont'd)

Linear interpolation

$$F(\zeta_1, \zeta_2, \zeta_3, \zeta_4) = F_1\zeta_1 + F_2\zeta_2 + F_3\zeta_3 + F_4\zeta_4 = F_i\zeta_i$$

Make  $F = I, x, y, z$  above and collect as matrix relation:

$$\begin{bmatrix} 1 \\ x \\ y \\ z \end{bmatrix} = \begin{bmatrix} 1 & 1 & 1 & 1 \\ x_1 & x_2 & x_3 & x_4 \\ y_1 & y_2 & y_3 & y_4 \\ z_1 & z_2 & z_3 & z_4 \end{bmatrix} \begin{bmatrix} \zeta_1 \\ \zeta_2 \\ \zeta_3 \\ \zeta_4 \end{bmatrix}$$

Inverting

$$\begin{bmatrix} \zeta_1 \\ \zeta_2 \\ \zeta_3 \\ \zeta_4 \end{bmatrix} = \frac{1}{6V} \begin{bmatrix} 6V_1 & a_1 & b_1 & c_1 \\ 6V_2 & a_2 & b_2 & c_2 \\ 6V_3 & a_3 & b_3 & c_3 \\ 6V_4 & a_4 & b_4 & c_4 \end{bmatrix} \begin{bmatrix} 1 \\ x \\ y \\ z \end{bmatrix}$$

## Partial Derivatives

$$\frac{\partial x}{\partial \zeta_i} = x_i, \quad \frac{\partial y}{\partial \zeta_i} = y_i, \quad \frac{\partial z}{\partial \zeta_i} = z_i$$

$$6\mathcal{V} \frac{\partial \zeta_i}{\partial x} = a_i, \quad 6\mathcal{V} \frac{\partial \zeta_i}{\partial y} = b_i, \quad 6\mathcal{V} \frac{\partial \zeta_i}{\partial z} = c_i$$

For any differentiable function  $F(x,y,z)$  given over the element, use chain rule to get the Cartesian partials:

$$\frac{\partial F}{\partial x} = \frac{1}{6\mathcal{V}} \left( \frac{\partial F}{\partial \zeta_1} a_1 + \frac{\partial F}{\partial \zeta_2} a_2 + \frac{\partial F}{\partial \zeta_3} a_3 + \frac{\partial F}{\partial \zeta_4} a_4 \right) = \frac{1}{6\mathcal{V}} \frac{\partial F}{\partial \zeta_i} a_i$$

$$\frac{\partial F}{\partial y} = \frac{1}{6\mathcal{V}} \left( \frac{\partial F}{\partial \zeta_1} b_1 + \frac{\partial F}{\partial \zeta_2} b_2 + \frac{\partial F}{\partial \zeta_3} b_3 + \frac{\partial F}{\partial \zeta_4} b_4 \right) = \frac{1}{6\mathcal{V}} \frac{\partial F}{\partial \zeta_i} b_i$$

$$\frac{\partial F}{\partial z} = \frac{1}{6\mathcal{V}} \left( \frac{\partial F}{\partial \zeta_1} c_1 + \frac{\partial F}{\partial \zeta_2} c_2 + \frac{\partial F}{\partial \zeta_3} c_3 + \frac{\partial F}{\partial \zeta_4} c_4 \right) = \frac{1}{6\mathcal{V}} \frac{\partial F}{\partial \zeta_i} c_i$$

## Displacement Interpolation for Use in TPE Principle

$$\begin{bmatrix} u_x \\ u_y \\ u_z \end{bmatrix} = \begin{bmatrix} u_{x1} & u_{x2} & u_{x3} & u_{x4} \\ u_{y1} & u_{y2} & u_{y3} & u_{y4} \\ u_{z1} & u_{z2} & u_{z3} & u_{z4} \end{bmatrix} \begin{bmatrix} \zeta_1 \\ \zeta_2 \\ \zeta_3 \\ \zeta_4 \end{bmatrix}$$

Appending to the geometry definition one has the **iso-P definition** of the linear tetrahedron

$$\begin{bmatrix} 1 \\ x \\ y \\ z \\ u_x \\ u_y \\ u_z \end{bmatrix} = \begin{bmatrix} 1 & 1 & 1 & 1 \\ x_1 & x_2 & x_3 & x_4 \\ y_1 & y_2 & y_3 & y_4 \\ z_1 & z_2 & z_3 & z_4 \\ u_{x1} & u_{x2} & u_{x3} & u_{x4} \\ u_{y1} & u_{y2} & u_{y3} & u_{y4} \\ u_{z1} & u_{z2} & u_{z3} & u_{z4} \end{bmatrix} \begin{bmatrix} \zeta_1 \\ \zeta_2 \\ \zeta_3 \\ \zeta_4 \end{bmatrix}$$

## Strain-Displacement Relations

$$\mathbf{e} = \begin{bmatrix} e_{xx} \\ e_{yy} \\ e_{zz} \\ 2e_{xy} \\ 2e_{yz} \\ 2e_{zx} \end{bmatrix} = \begin{bmatrix} \partial/\partial x & 0 & 0 \\ 0 & \partial/\partial y & 0 \\ 0 & 0 & \partial/\partial z \\ \partial/\partial y & \partial/\partial x & 0 \\ 0 & \partial/\partial z & \partial/\partial y \\ \partial/\partial z & 0 & \partial/\partial x \end{bmatrix} \begin{bmatrix} u_x \\ u_y \\ u_z \end{bmatrix} = \mathbf{D} \vec{\mathbf{u}}.$$

Inserting the displacement field interpolation one gets

$$\mathbf{e} = \mathbf{B} \mathbf{u}^e$$

## Strain-Displacement Relations (cont'd)

If the element node displacement is arranged component-wise:

$$\mathbf{u}^e = [u_{x1} \quad u_{x2} \quad u_{x3} \quad u_{x4} \quad u_{y1} \quad u_{y2} \quad \cdots \quad u_{z4}]^T$$

$$\mathbf{B} = \frac{1}{6V} \begin{bmatrix} a_1 & a_2 & a_3 & a_4 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & b_1 & b_2 & b_3 & b_4 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & c_1 & c_2 & c_3 & c_4 \\ b_1 & b_2 & b_3 & b_4 & a_1 & a_2 & a_3 & a_4 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & c_1 & c_2 & c_3 & c_4 & b_1 & b_2 & b_3 & b_4 \\ c_1 & c_2 & c_3 & c_4 & 0 & 0 & 0 & 0 & a_1 & a_2 & a_3 & a_4 \end{bmatrix}$$

If the element node displacement is arranged node-wise:

$$\mathbf{u}^e = [u_{x1} \quad u_{y1} \quad u_{z1} \quad u_{x2} \quad u_{y2} \quad u_{z2} \quad \cdots \quad u_{z4}]^T$$

$$\mathbf{B} = \frac{1}{6V} \begin{bmatrix} a_1 & 0 & 0 & a_2 & 0 & 0 & a_3 & 0 & 0 & a_4 & 0 & 0 \\ 0 & b_1 & 0 & 0 & b_2 & 0 & 0 & b_3 & 0 & 0 & b_4 & 0 \\ 0 & 0 & c_1 & 0 & 0 & c_2 & 0 & 0 & c_3 & 0 & 0 & c_4 \\ b_1 & a_1 & 0 & b_2 & a_2 & 0 & b_3 & a_3 & 0 & b_4 & a_4 & 0 \\ 0 & c_1 & b_1 & 0 & c_2 & b_2 & 0 & c_3 & b_3 & 0 & c_4 & b_4 \\ c_1 & 0 & a_1 & c_2 & 0 & a_2 & c_3 & 0 & a_3 & c_4 & 0 & a_4 \end{bmatrix}$$

## Stress-Strain (Constitutive) Equations

$$\sigma_{ij} = E_{ijkl} e_{kl}$$

$$\begin{aligned} \boldsymbol{\sigma} &= [\sigma_{11} \quad \sigma_{22} \quad \sigma_{33} \quad \sigma_{12} \quad \sigma_{23} \quad \sigma_{31}]^T = \\ &= [\sigma_{xx} \quad \sigma_{yy} \quad \sigma_{zz} \quad \sigma_{xy} \quad \sigma_{yz} \quad \sigma_{zx}]^T \end{aligned}$$

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \sigma_{xy} \\ \sigma_{yz} \\ \sigma_{zx} \end{bmatrix} = \begin{bmatrix} E_{11} & E_{12} & E_{13} & E_{14} & E_{15} & E_{16} \\ & E_{22} & E_{23} & E_{24} & E_{25} & E_{26} \\ & & E_{33} & E_{34} & E_{35} & E_{36} \\ & & & E_{44} & E_{45} & E_{46} \\ & & & & E_{55} & E_{56} \\ & & & & & E_{66} \end{bmatrix} \begin{bmatrix} e_{xx} \\ e_{yy} \\ e_{zz} \\ 2e_{xy} \\ 2e_{yz} \\ 2e_{zx} \end{bmatrix}$$

symm

$$\boldsymbol{\sigma} = \mathbf{E} \mathbf{e}$$

## Element Stiffness Matrix from TPE

$$\mathbf{K}^e = \int_{V^e} \mathbf{B}^T \mathbf{E} \mathbf{B} dV$$

If the elasticity matrix  $\mathbf{E}$  is constant over element:

$$\mathbf{K}^e = \mathcal{V} \mathbf{B}^T \mathbf{E} \mathbf{B}$$

This stiffness matrix is 12 x 12, and has rank 6  
(assuming  $\mathbf{E}$  and  $\mathbf{B}$  have full rank)

## Element Consistent Node Force Vector for Body Force Field

$$\mathbf{b} = \begin{bmatrix} b_x \\ b_y \\ b_z \end{bmatrix} \quad \begin{bmatrix} u_x \\ u_y \\ u_z \end{bmatrix} = \mathbf{N} \mathbf{u}^e$$

The TPE principle gives

$$\mathbf{f}^e = \int_{V^e} \mathbf{N}^T \mathbf{b} dV$$

## Element Consistent Node Force Vector for Body Force Field (cont'd)

If the node displacement vector is arranged component-wise:

$$\mathbf{N} = \begin{bmatrix} \zeta_1 & \zeta_2 & \zeta_3 & \zeta_4 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & \zeta_1 & \zeta_2 & \zeta_3 & \zeta_4 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & \zeta_1 & \zeta_2 & \zeta_3 & \zeta_4 \end{bmatrix}$$

If the node displacement vector is arranged node-wise:

$$\mathbf{N} = \begin{bmatrix} \zeta_1 & 0 & 0 & \zeta_2 & 0 & 0 & \zeta_3 & 0 & 0 & \zeta_4 & 0 & 0 \\ 0 & \zeta_1 & 0 & 0 & \zeta_2 & 0 & 0 & \zeta_3 & 0 & 0 & \zeta_4 & 0 \\ 0 & 0 & \zeta_1 & 0 & 0 & \zeta_2 & 0 & 0 & \zeta_3 & 0 & 0 & \zeta_4 \end{bmatrix}$$

## Useful Exact Volume-Integration Formulas in Terms of Tetrahedron Coordinates

Linear integrand

$$\int_{V^e} \zeta_i dV = \mathcal{V}/4$$

Quadratic integrand

$$\int_{V^e} \zeta_i \zeta_j dV = \begin{cases} \mathcal{V}/10 & \text{if } i = j \\ \mathcal{V}/20 & \text{if } i \neq j \end{cases}$$

General formula for a monomial term

$$\int_{V^e} \zeta_1^i \zeta_2^j \zeta_3^k \zeta_4^\ell dV = \frac{i! j! k! \ell!}{(i + j + k + \ell + 3)!} 6\mathcal{V}$$

## Tetr4 Stiffness Matrix Module

```

IsoTetr4Stiffness[ncoor_,Emat_,{ },options_]:= Module[{i,n=4,nf=12,
  k,c,w,Jdet,zetalist,xyzlist,numer,Bx,By,Bz,Be,Ke},
  If [Length[options]>0, numer=options[[1]]];
  xyzlist={Table[ncoor[[i,1]],{i,n}],Table[ncoor[[i,2]],{i,n}],
    Table[ncoor[[i,3]],{i,n}]}; Ke=Table[0,{nf},{nf}];
  {Bx,By,Bz,Jdet}=IsoTetr4ShapeFunCarDer[xyzlist,{ },numer];
  Be={Flatten[Table[{Bx[[i]],0, 0 },{i,n}]],
    Flatten[Table[{0, By[[i]],0 },{i,n}]],
    Flatten[Table[{0, 0, Bz[[i]]},{i,n}]],
    Flatten[Table[{By[[i]],Bx[[i]],0 },{i,n}]],
    Flatten[Table[{0, Bz[[i]],By[[i]]},{i,n}]],
    Flatten[Table[{Bz[[i]],0, Bx[[i]]},{i,n}]]];
  Ke=(Jdet/6)*Transpose[Be].(Emat.Be);
  If [!numer,Ke=Simplify[Ke]]; Return[Ke]
];

```

## Shape Function Cartesian Derivatives Module

```

IsoTetr4ShapeFunCarDer[{xn_, yn_, zn_}, zetalist_, numer_] :=
Module[{dNz1, dNz2, dNz3, dNz4, Jmat, J11, J12, J13, J14,
  J21, J22, J23, J24, J31, J32, J33, J34, Jinv, Jdet, Bx, By, Bz},
  {dNz1, dNz2, dNz3, dNz4} = {{1, 0, 0, 0}, {0, 1, 0, 0}, {0, 0, 1, 0}, {0, 0, 0, 1}};
  J11 = dNz1.xn; J12 = dNz2.xn; J13 = dNz3.xn; J14 = dNz4.xn;
  J21 = dNz1.yn; J22 = dNz2.yn; J23 = dNz3.yn; J24 = dNz4.yn;
  J31 = dNz1.zn; J32 = dNz2.zn; J33 = dNz3.zn; J34 = dNz4.zn;
  Jmat = {{1, 1, 1, 1}, {J11, J12, J13, J14},
    {J21, J22, J23, J24}, {J31, J32, J33, J34}};
  Jdet = (J13*J22 - J12*J23 + J14*J23 - J14*J22 + J12*J24 - J13*J24)*J31 -
    (J13*J21 - J11*J23 + J14*J23 - J14*J21 + J11*J24 - J13*J24)*J32 +
    (J12*J21 - J11*J22 + J14*J22 - J14*J21 + J11*J24 - J12*J24)*J33 -
    (J12*J21 - J11*J22 + J13*J22 - J13*J21 + J11*J23 - J12*J23)*J34;
  Jinv = {{J22*(J34 - J33) - J23*(J34 - J32) + J24*(J33 - J32),
    -J12*(J34 - J33) + J13*(J34 - J32) - J14*(J33 - J32),
    J12*(J24 - J23) - J13*(J24 - J22) + J14*(J23 - J22)},
    {-J21*(J34 - J33) + J23*(J34 - J31) - J24*(J33 - J31),
    J11*(J34 - J33) - J13*(J34 - J31) + J14*(J33 - J31),
    -J11*(J24 - J23) + J13*(J24 - J21) - J14*(J23 - J21)},
    {J21*(J34 - J32) - J22*(J34 - J31) + J24*(J32 - J31),
    -J11*(J34 - J32) + J12*(J34 - J31) - J14*(J32 - J31),
    J11*(J24 - J22) - J12*(J24 - J21) + J14*(J22 - J21)},
    {-J21*(J33 - J32) + J22*(J33 - J31) - J23*(J32 - J31),
    J11*(J33 - J32) - J12*(J33 - J31) + J13*(J32 - J31),
    -J11*(J23 - J22) + J12*(J23 - J21) - J13*(J22 - J21)}};
  {Bx, By, Bz} = Transpose[Jinv].{dNz1, dNz2, dNz3, dNz4}/Jdet;
  Return[{Bx, By, Bz, Jdet}]
];

```

# Testing the Stiffness Module

Advanced FEM

```

ClearAll[Em,v]; Em=96; v=1/3;
Emat=Em/((1+v)*(1-2*v))*{{1-v,v,v,0,0,0},
  {v,1-v,v,0,0,0},{v,v,1-v,0,0,0},{0,0,0,1/2-v,0,0},
  {0,0,0,1/2-v,0},{0,0,0,0,0,1/2-v}};
Print["Emat=",Emat//MatrixForm];
ncoor={{2,3,4},{6,3,2},{2,5,1},{4,3,6}};
Ke=IsoTetr4Stiffness[ncoor,Emat,{},{False}];
Print["Ke=",Ke//MatrixForm];
Print["eigs of Ke=",Chop[Eigenvalues[N[Ke]]]];
    
```

$$\text{Emat} = \begin{pmatrix} 144 & 72 & 72 & 0 & 0 & 0 \\ 72 & 144 & 72 & 0 & 0 & 0 \\ 72 & 72 & 144 & 0 & 0 & 0 \\ 0 & 0 & 0 & 36 & 0 & 0 \\ 0 & 0 & 0 & 0 & 36 & 0 \\ 0 & 0 & 0 & 0 & 0 & 36 \end{pmatrix}$$

$$\text{Ke} = \begin{pmatrix} 149 & 108 & 24 & -1 & 6 & 12 & -54 & -48 & 0 & -94 & -66 & -36 \\ 108 & 344 & 54 & -24 & 104 & 42 & -24 & -216 & -12 & -60 & -232 & -84 \\ 24 & 54 & 113 & 0 & 30 & 35 & 0 & -24 & -54 & -24 & -60 & -94 \\ -1 & -24 & 0 & 29 & -18 & -12 & -18 & 24 & 0 & -10 & 18 & 12 \\ 6 & 104 & 30 & -18 & 44 & 18 & 12 & -72 & -12 & 0 & -76 & -36 \\ 12 & 42 & 35 & -12 & 18 & 29 & 0 & -24 & -18 & 0 & -36 & -46 \\ -54 & -24 & 0 & -18 & 12 & 0 & 36 & 0 & 0 & 36 & 12 & 0 \\ -48 & -216 & -24 & 24 & -72 & -24 & 0 & 144 & 0 & 24 & 144 & 48 \\ 0 & -12 & -54 & 0 & -12 & -18 & 0 & 0 & 36 & 0 & 24 & 36 \\ -94 & -60 & -24 & -10 & 0 & 0 & 36 & 24 & 0 & 68 & 36 & 24 \\ -66 & -232 & -60 & 18 & -76 & -36 & 12 & 144 & 24 & 36 & 164 & 72 \\ -36 & -84 & -94 & 12 & -36 & -46 & 0 & 48 & 36 & 24 & 72 & 104 \end{pmatrix}$$

eigs of Ke = {777.175, 201.363, 197.273, 42.9431, 21.3643, 19.8821, 0, 0, 0, 0, 0, 0}